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**Analytical Predesign of a Pitch Controller for
Rotational Speed Regulation in the Time and
Frequency Domain of a Multi-Megawatt Wind Turbine
Based on Competing Criteria**

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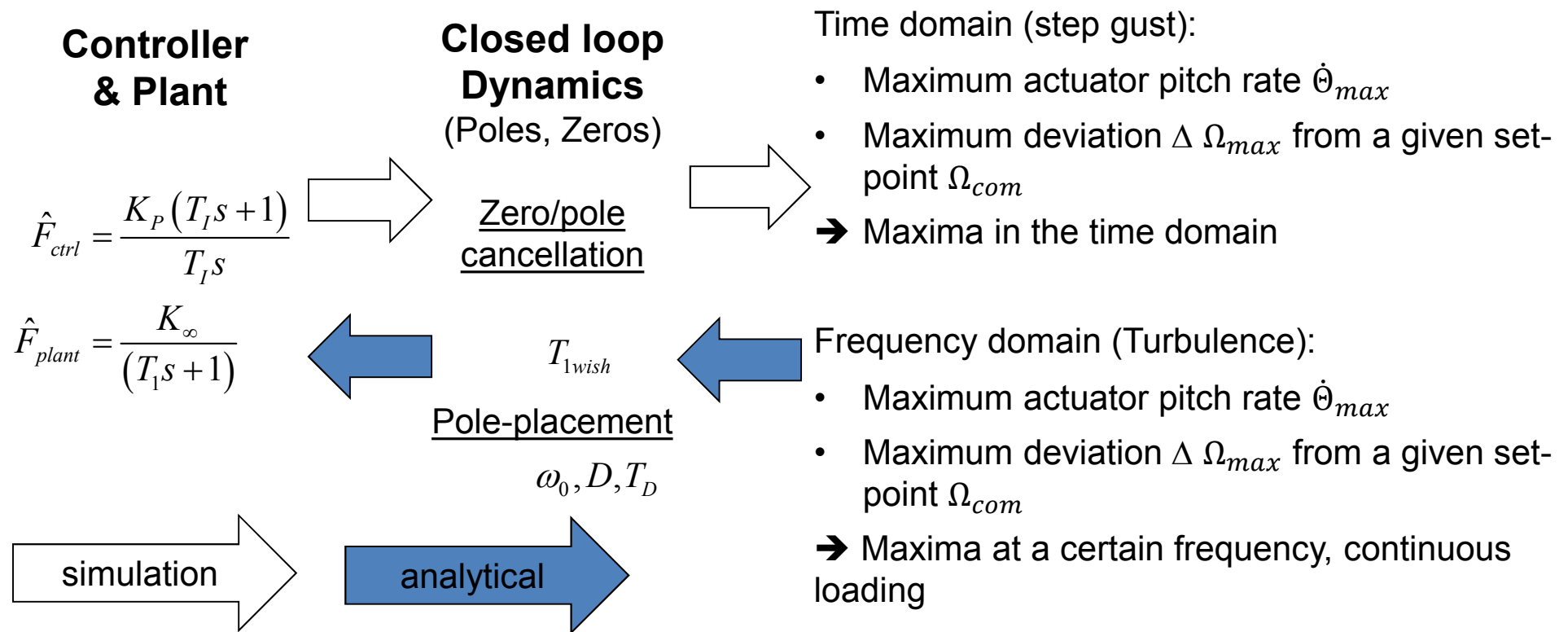
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Wolfenbüttel, Germany



Introduction

- Rotational speed Ω set-point tracking of variable-pitch wind turbines often requires the tuning of PI-controller

Criteria (disturbance reaction)



Introduction

- It is not about
 - a comparison of the two methods
 - the perfect PI-controller tuning
- It is about two methods for PI-controller tuning
 - One for extreme events like step gust in the time domain
 - One for continuous loading in the frequency domain
- **Objective:**
 - analytical advice
 - better physical understanding
 - good trade off
 - show what is possible and what is not possible
- **Master objective:** find easy to handle equations for a “paper and pencil” method usable for controller predesign



Outline

- Wind turbine model (linear)
- Disturbance models
- Closed-loop control loop
 - zero/pole cancellation
 - pole-placement
- Nonlinear simulation
- Summary



Wind turbine model

- Based on the law of conservation of angular momentum
- Transfer function: pitch $\delta\Theta \rightarrow$ rotational speed $\delta\Omega$

$$\hat{F}_{\Omega\Theta} = \frac{-\frac{\Omega_0}{P_0} \frac{\partial P}{\partial \Theta}}{\frac{I\Omega_0^2}{P_0}s + 1} = \frac{K_{\infty\Theta}}{(T_1s + 1)}$$

- Transfer function: wind $\delta V_w \rightarrow$ rotational speed $\delta\Omega$

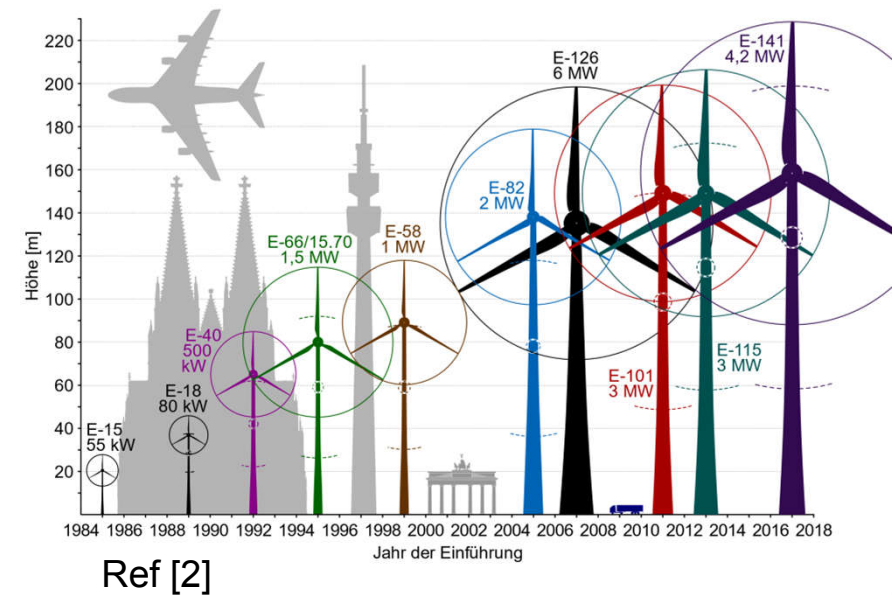
$$\hat{F}_{\Omega V_w} = \frac{-\frac{\Omega_0}{P_0} \frac{\partial P}{\partial V_w}}{\frac{I\Omega_0^2}{P_0}s + 1} = \frac{K_{\infty V_w}}{(T_1s + 1)}$$

- Steady-state rotational speed $\Omega_0 = 12,1 \text{ rpm}$
- Steady-state power $P_0 = 5,3 \text{ MW}$
- Moment of inertia $I = 43,7 \text{ MKgm}^2$
- Power-pitch sensitivity $\frac{\partial P}{\partial \Theta} = -25,0 \frac{\text{MW}}{\text{rad}}$
- Power-wind sensitivity $\frac{\partial P}{\partial V_w} = -0,6 \frac{\text{MW}}{\text{m/s}}$



Wind turbine model: NREL 5 MW

Description	Value
Rated Power (MW)	5.0
Rotor diameter (m)	126.0
Hub Height (m)	90.0
Rated Rotor speed (rpm)	12.1
Cut-in, Rated, Cut-out speed (m/s)	3.0, 11.4, 25.0



NREL: National Renewable Energy Laboratory (Colorado, USA)



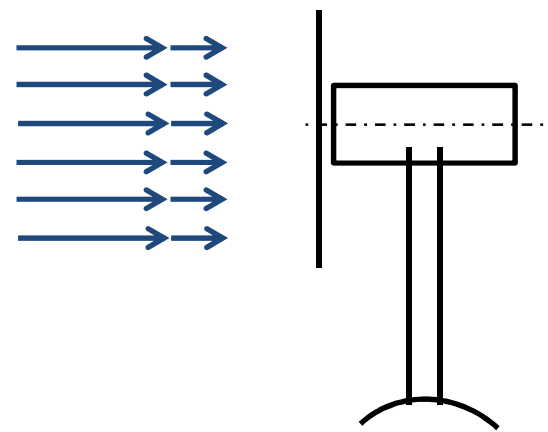
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Disturbance models

- A step disturbance is a strong simplification of real gusts, since a step is not possible in nature
- First conservative (compared to a 1-cos gust) estimate
 - Max. disturbance reaction $\Delta\Omega_{\max}$
 - Max. actuator pitch rate $\dot{\Theta}_{\max}$
- Suitable for the rapid preliminary analytical design
- Considered: $\Delta V_W = 1\text{ m/s}$ at $V_W = 12\text{ m/s}$
- Linear system => scalable



Disturbance models

- 1-D approximation of real (measured) gust power spectral (PSD) density
- For simulation purpose zero mean Gaussian white noise is shaped through a Dryden form filter

$$\hat{F}_{V_w} = \frac{\sigma_{V_w} \sqrt{2T_{V_w}}}{\underbrace{1 + T_{V_w} s}_{\text{Dryden}}}$$

standard deviation (turbulence strength): $\sigma_{V_w} = 1,8 \text{ m / s}$

characteristic time constant: $T_{V_w} = 7,7 \text{ s}$

wind speed: $V_w = 12 \text{ m / s}$

- These values are tuned to meet the 3D Turbulence characteristics in the nonlinear simulation (FAST) at hub height
- low-pass characteristic → high frequency components are attenuated → acceptably frequency response approximation for analytical predesign



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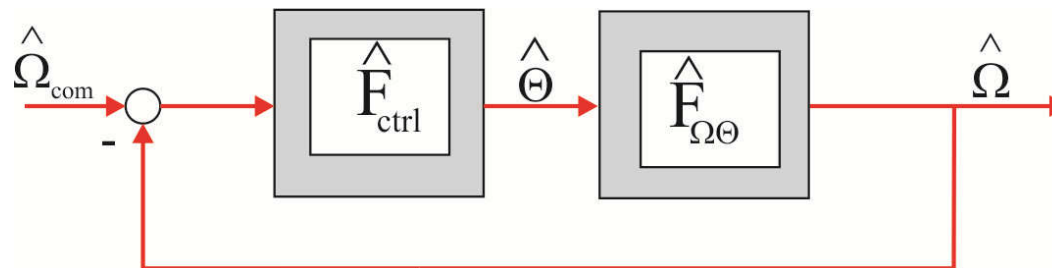


Closed-loop control: zero/pole cancellation reference reaction

PT₁

$$\hat{F}_o = \underbrace{\frac{K_P (T_I s + 1)}{T_I s}}_{\text{Controller}} \underbrace{\frac{K_{\infty\Theta}}{(T_1 s + 1)}}_{\text{Plant}} \Rightarrow \hat{F}_{\Omega_{com} FZ} = \frac{1}{\underbrace{\frac{T_1}{K_{\infty\Theta} K_P}}_{T_{1wish}} s + 1}$$

$$\begin{aligned} T_I &= T_1 \\ \Rightarrow K_P &= \frac{T_1}{T_{1wish} K_{\infty\Theta}} \end{aligned}$$

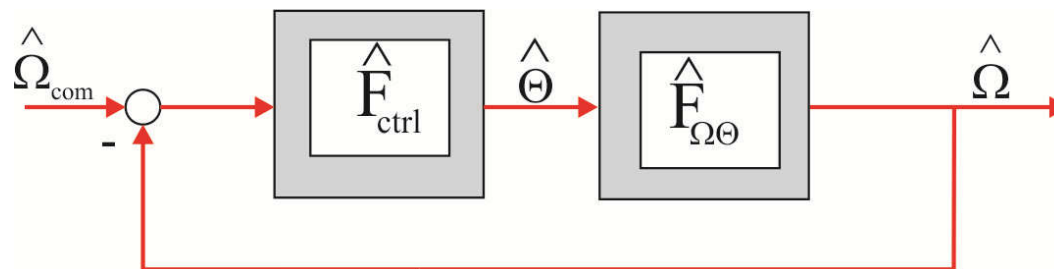


Closed-loop control: pole-placement reference reaction

PDT₂

$$\hat{F}_o = \underbrace{\frac{K_P (T_I s + 1)}{T_I s}}_{\text{Controller}} \underbrace{\frac{K_{\infty\Theta}}{(T_I s + 1)}}_{\text{Plant}} \Rightarrow \hat{F}_{\Omega\Omega_{com} Pol} = \underbrace{\frac{\omega_0^2 (T_D s + 1)}{s^2 + 2D\omega_0 s + \omega_0^2}}_{\text{reference system}}$$

$$\Rightarrow K_P = \frac{2D\omega_0 T_1 - 1}{K_{\infty\Theta}}; \quad T_I = T_D = \frac{2D\omega_0 T_1 - 1}{\omega_0^2 T_1}$$

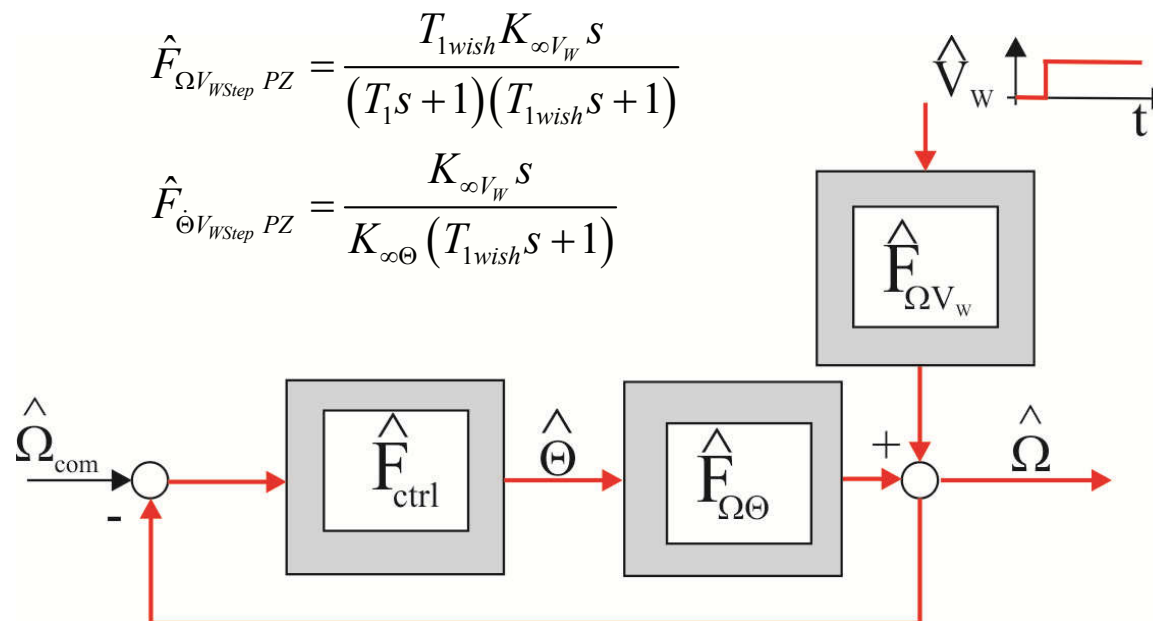


To prevent all-pass behavior (initial and steady state response a different) :

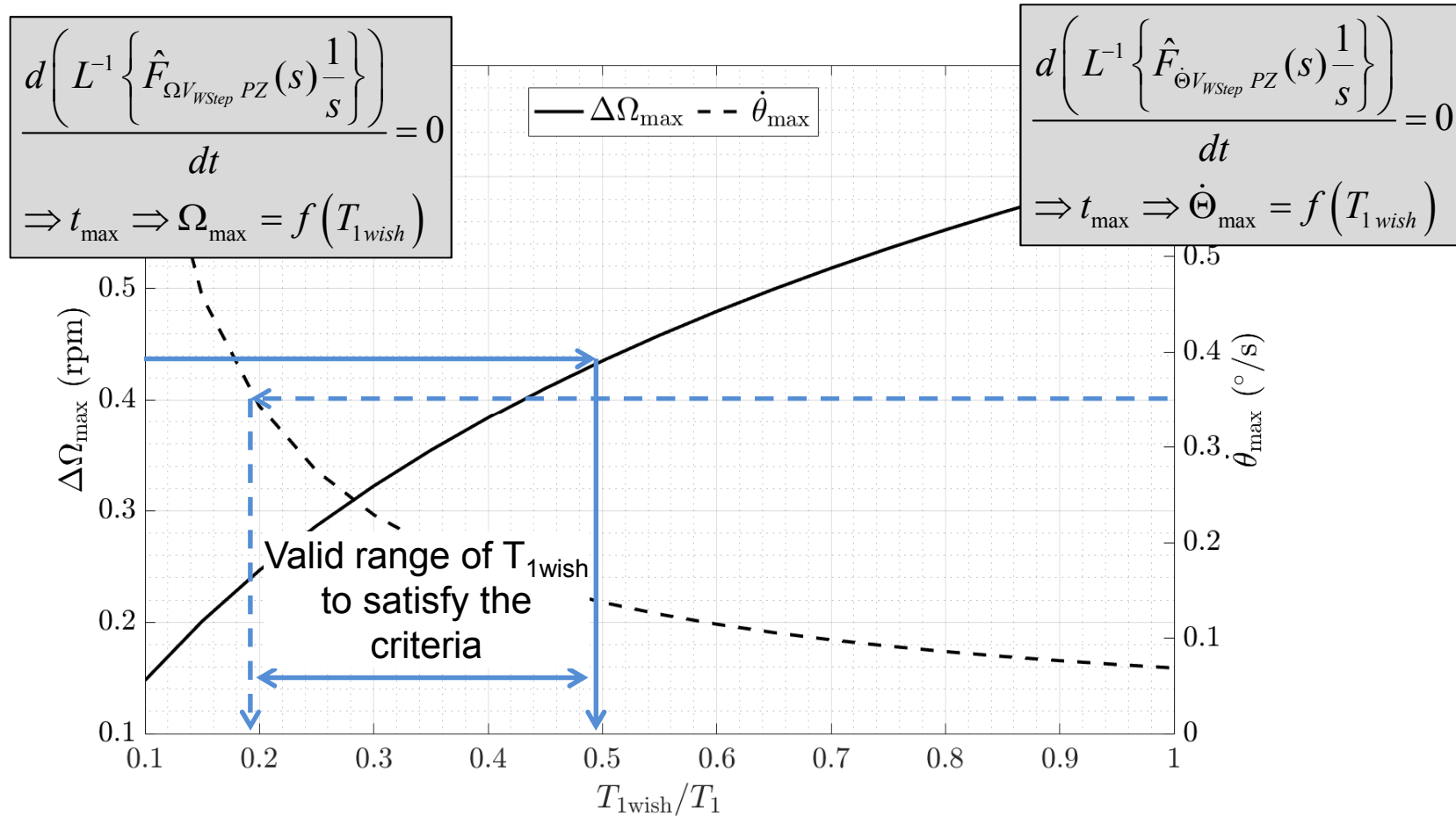
$$T_D = f(D, \omega_0, T_1) \geq 0 \Rightarrow \omega_{0min} \geq (2DT_1)^{-1}$$



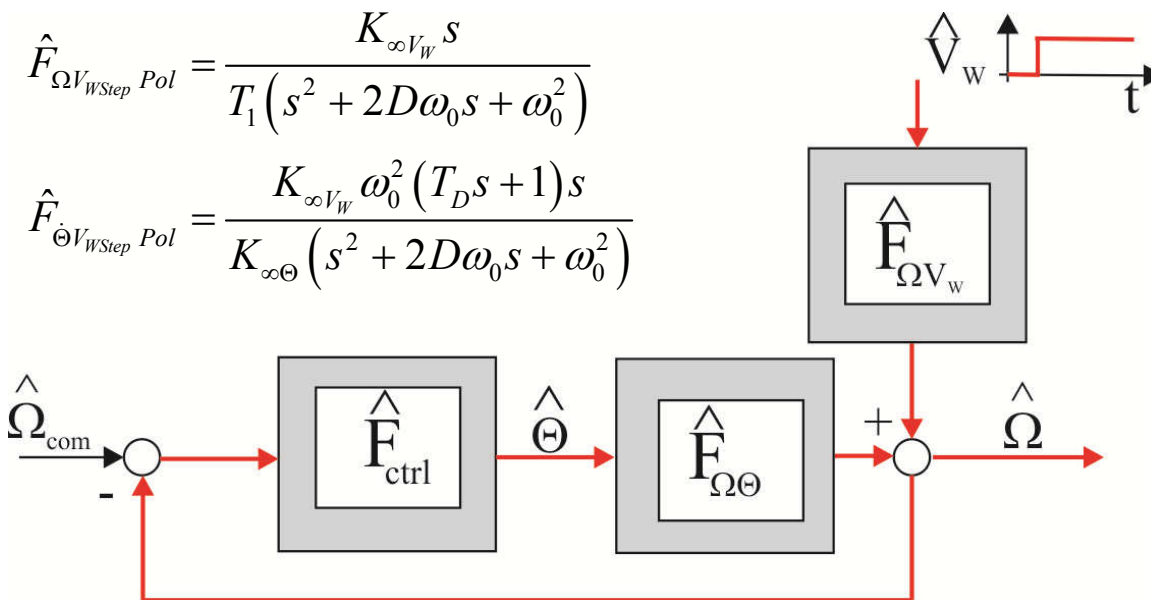
Closed-loop control: zero/pole cancellation disturbance reaction (time domain)



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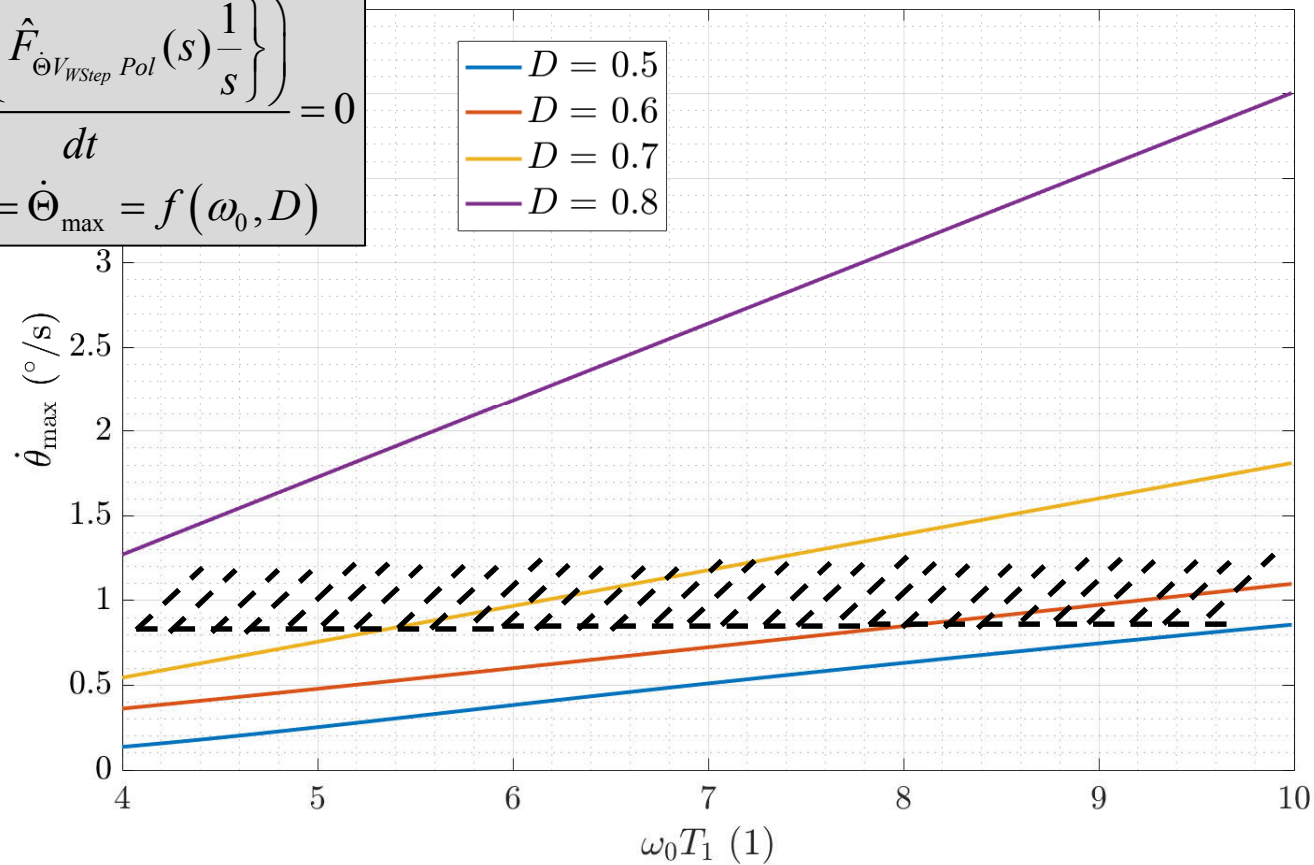
Closed-loop control: pole-placement disturbance reaction (time domain)



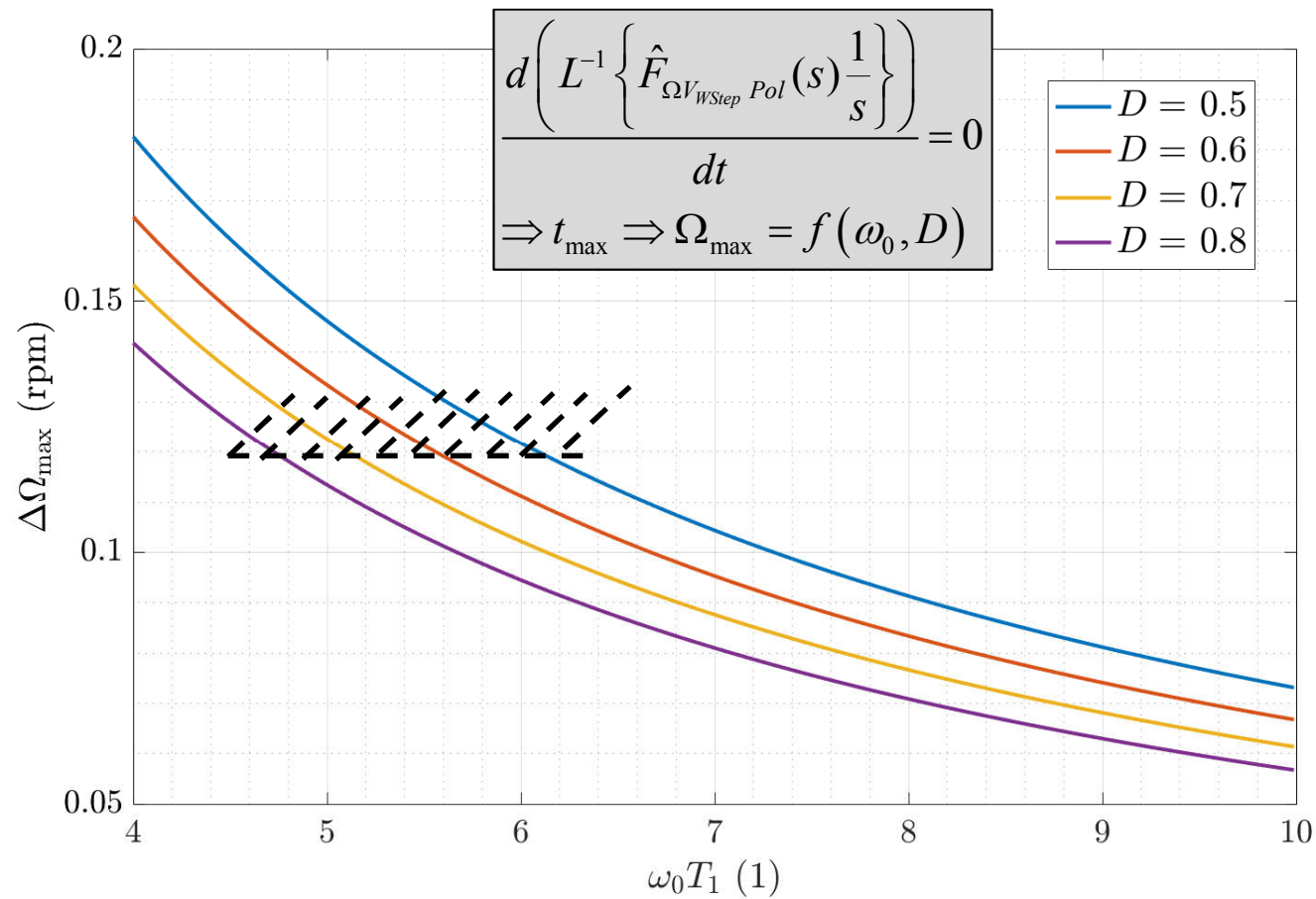
Closed-loop control: pole-placement disturbance reaction (time domain)

$$\frac{d \left(L^{-1} \left\{ \hat{F}_{\dot{\Theta}_{WStep} Pol} (s) \frac{1}{s} \right\} \right)}{dt} = 0$$

$$\Rightarrow t_{\max} = \dot{\Theta}_{\max} = f(\omega_0, D)$$



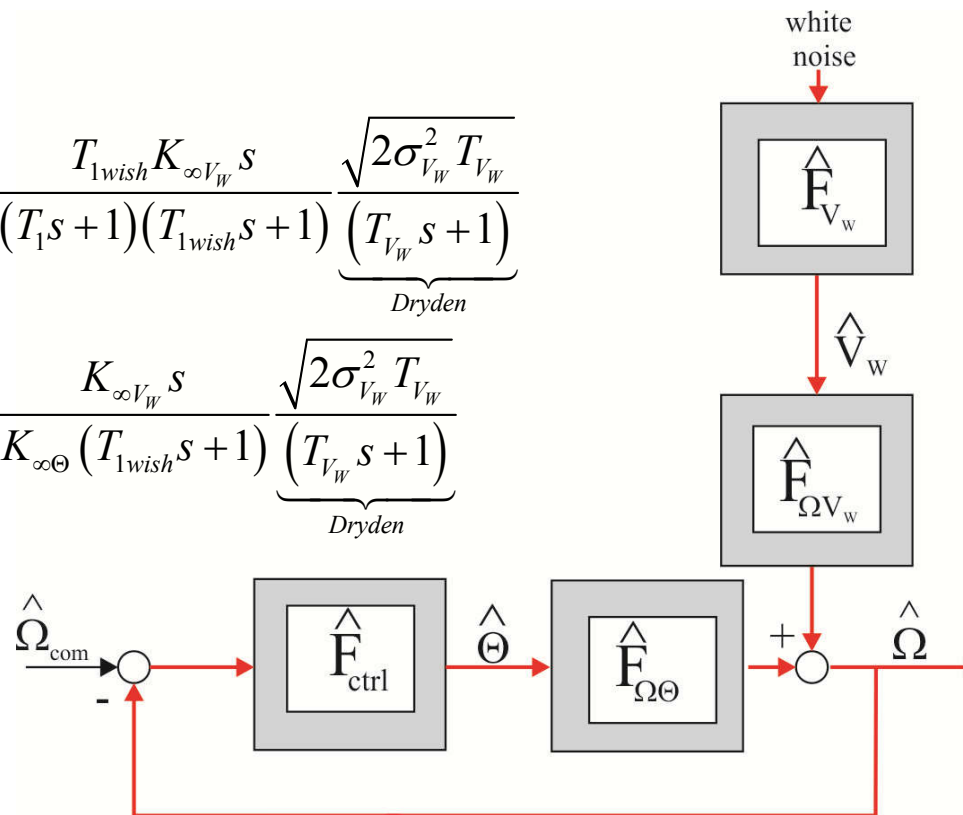
Closed-loop control: pole-placement disturbance reaction (time domain)



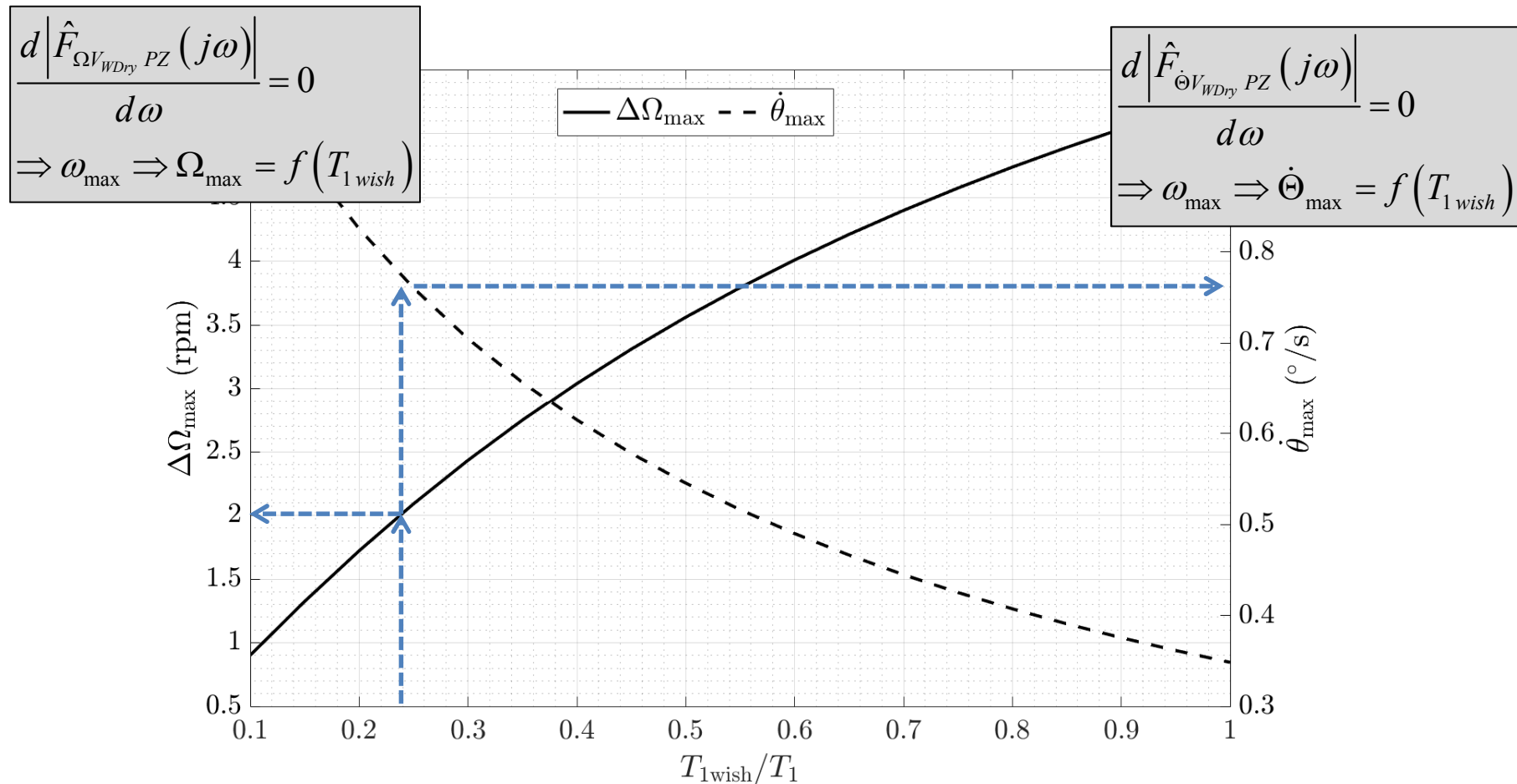
Closed-loop control: zero/pole cancellation disturbance reaction (frequency domain)

$$\hat{F}_{\Omega V_{WDr}, PZ} = \frac{T_{1wish} K_{\infty V_W} s}{(T_1 s + 1)(T_{1wish} s + 1)} \underbrace{\frac{\sqrt{2\sigma_{V_W}^2 T_{V_W}}}{(T_{V_W} s + 1)}}_{Dryden}$$

$$\hat{F}_{\dot{\Theta} V_{WDr}, PZ} = \frac{K_{\infty V_W} s}{K_{\infty \Theta} (T_{1wish} s + 1)} \underbrace{\frac{\sqrt{2\sigma_{V_W}^2 T_{V_W}}}{(T_{V_W} s + 1)}}_{Dryden}$$



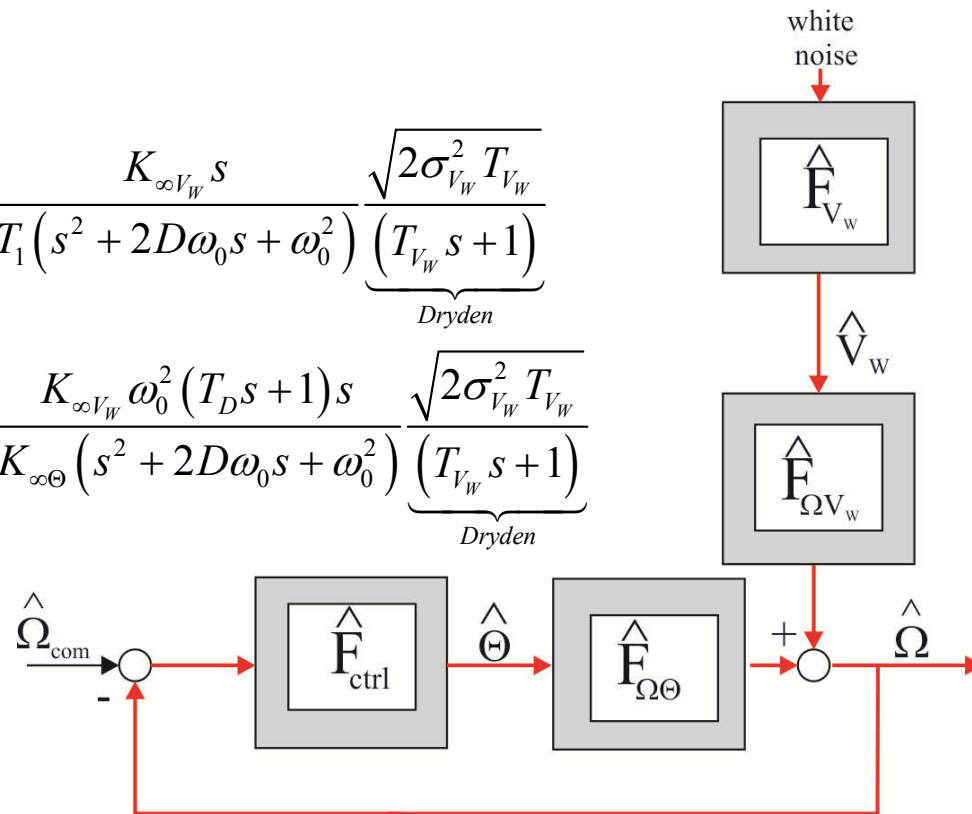
Closed-loop control: zero/pole cancellation disturbance reaction (frequency domain)



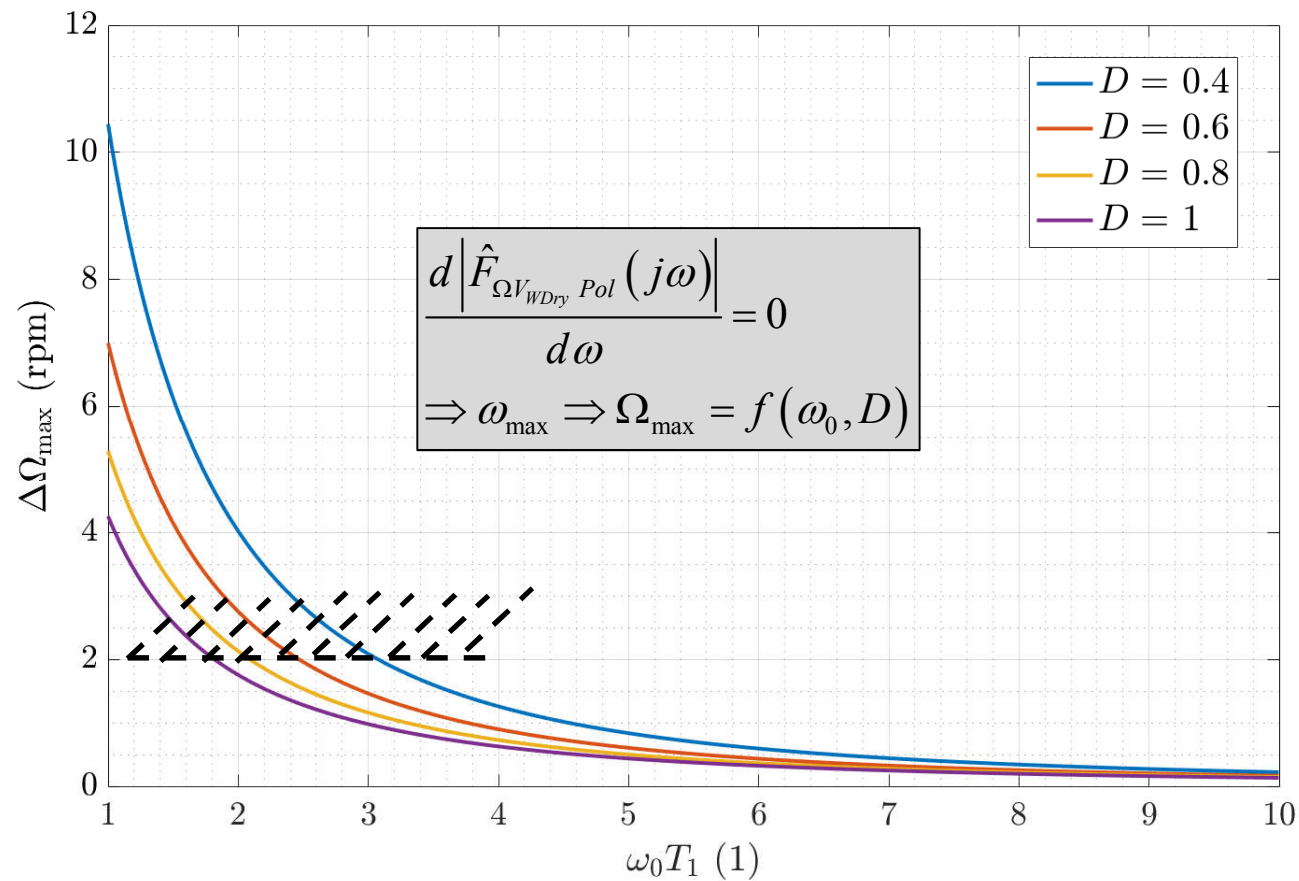
Closed-loop control: pole-placement disturbance reaction (frequency domain)

$$\hat{F}_{\Omega V_{WDry} Pol} = \frac{K_{\infty V_W} s}{T_1 (s^2 + 2D\omega_0 s + \omega_0^2)} \underbrace{\frac{\sqrt{2\sigma_{V_W}^2 T_{V_W}}}{(T_{V_W} s + 1)}}_{Dryden}$$

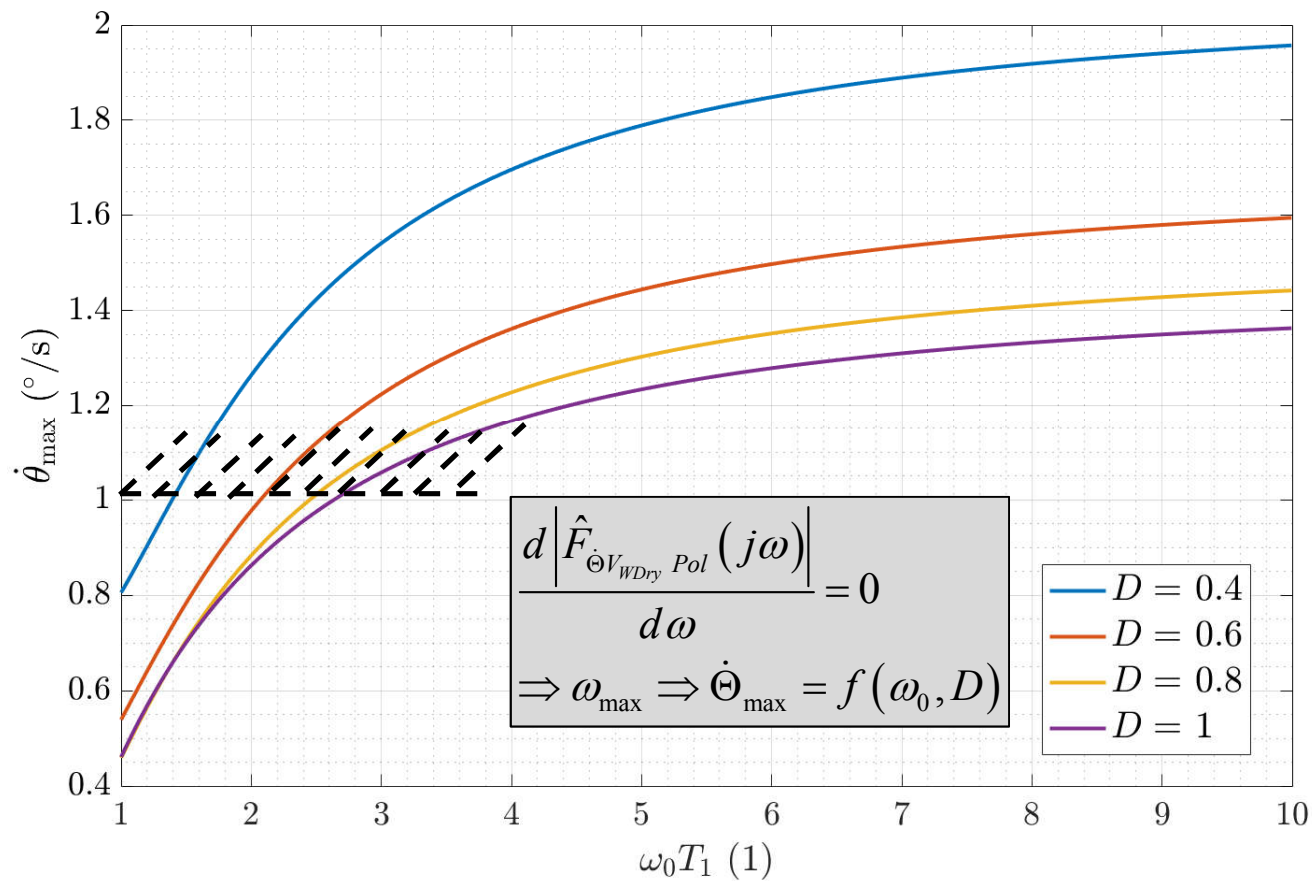
$$\hat{F}_{\Theta V_{WDry} Pol} = \frac{K_{\infty V_W} \omega_0^2 (T_D s + 1) s}{K_{\infty \Theta} (s^2 + 2D\omega_0 s + \omega_0^2)} \underbrace{\frac{\sqrt{2\sigma_{V_W}^2 T_{V_W}}}{(T_{V_W} s + 1)}}_{Dryden}$$



Closed-loop control: pole-placement disturbance reaction (frequency domain)



Closed-loop control: pole-placement disturbance reaction (frequency domain)



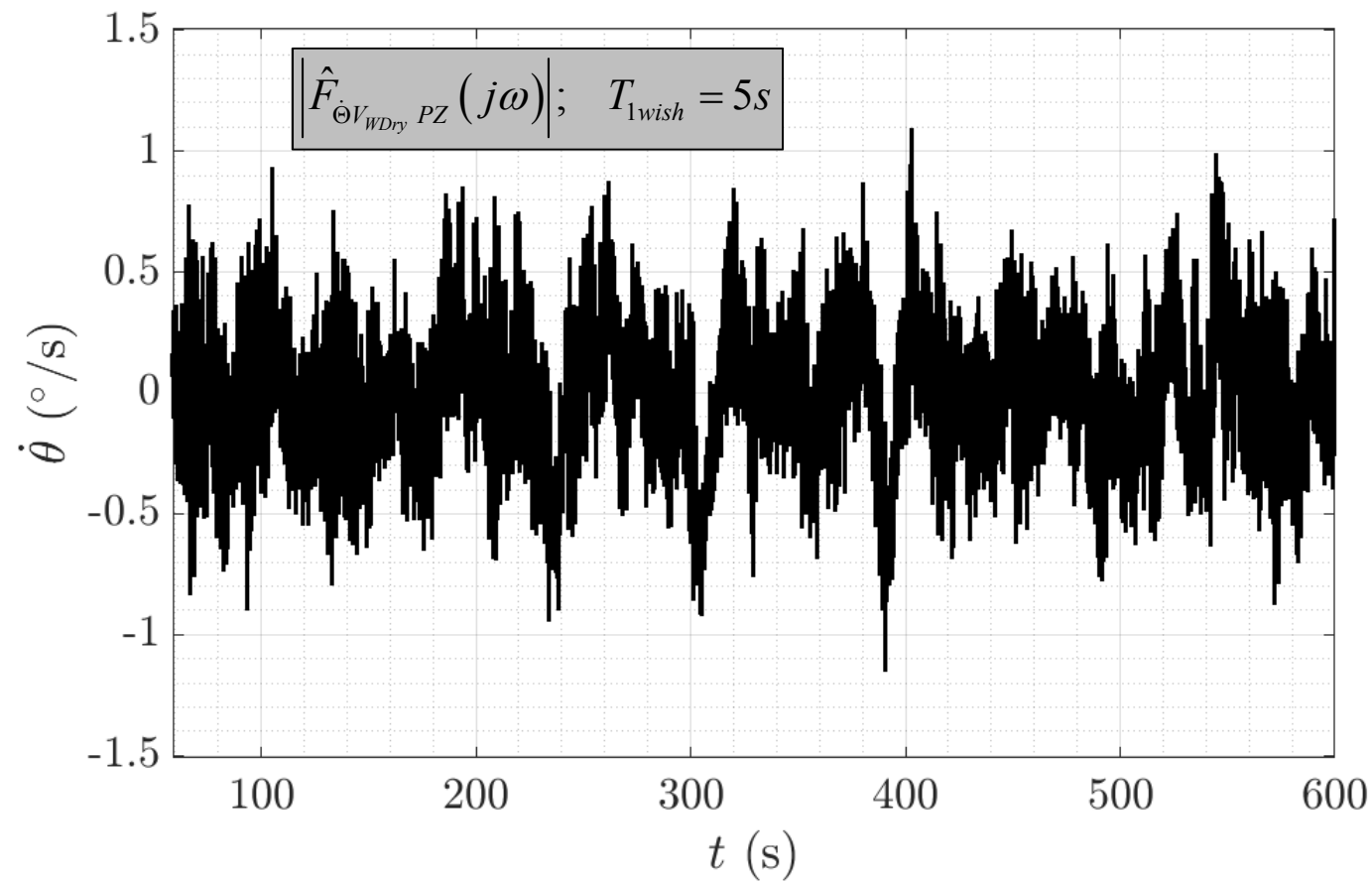
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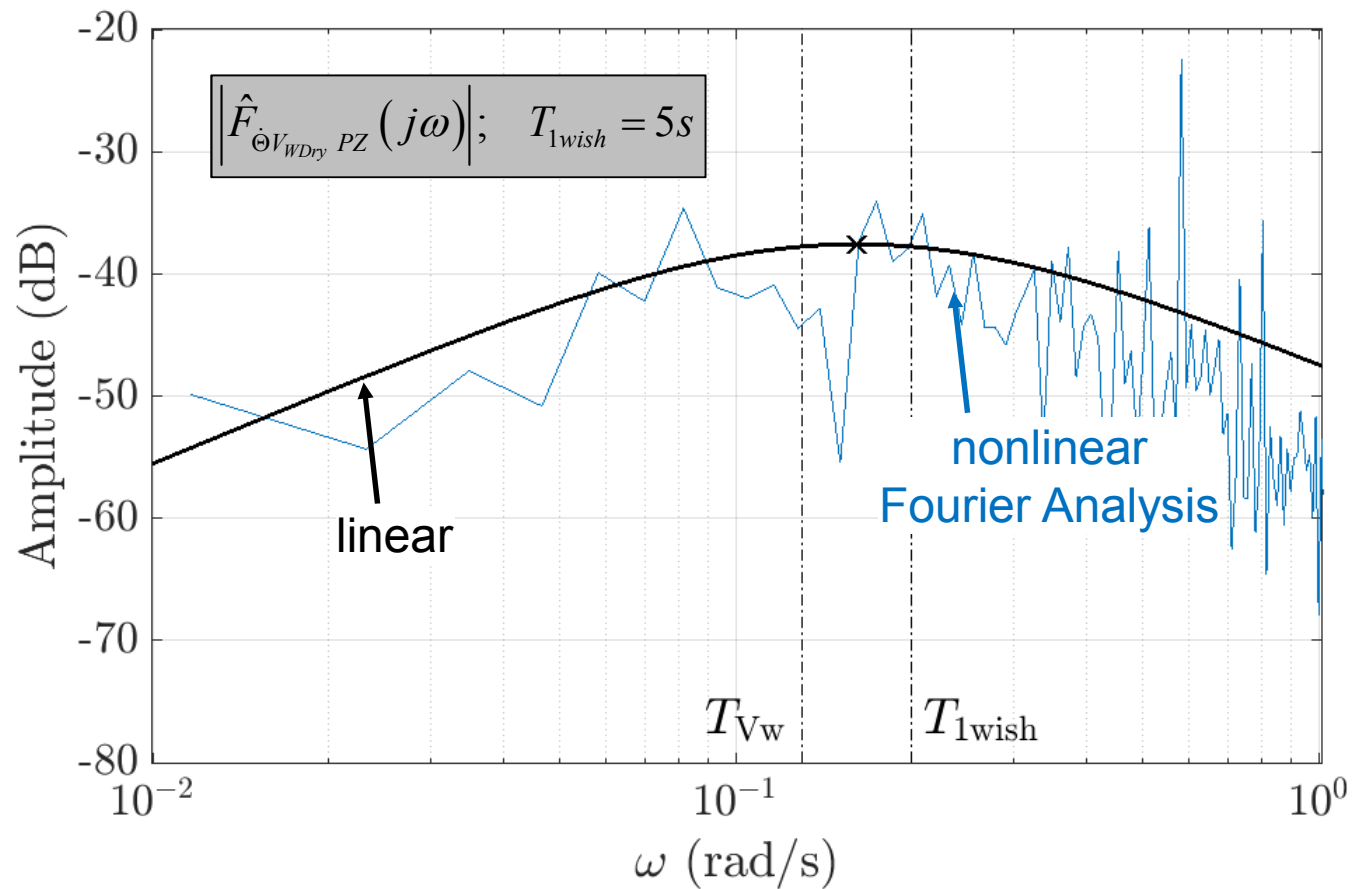
Nonlinear simulation (FAST)

zero/pole cancellation disturbance reaction (frequency domain)



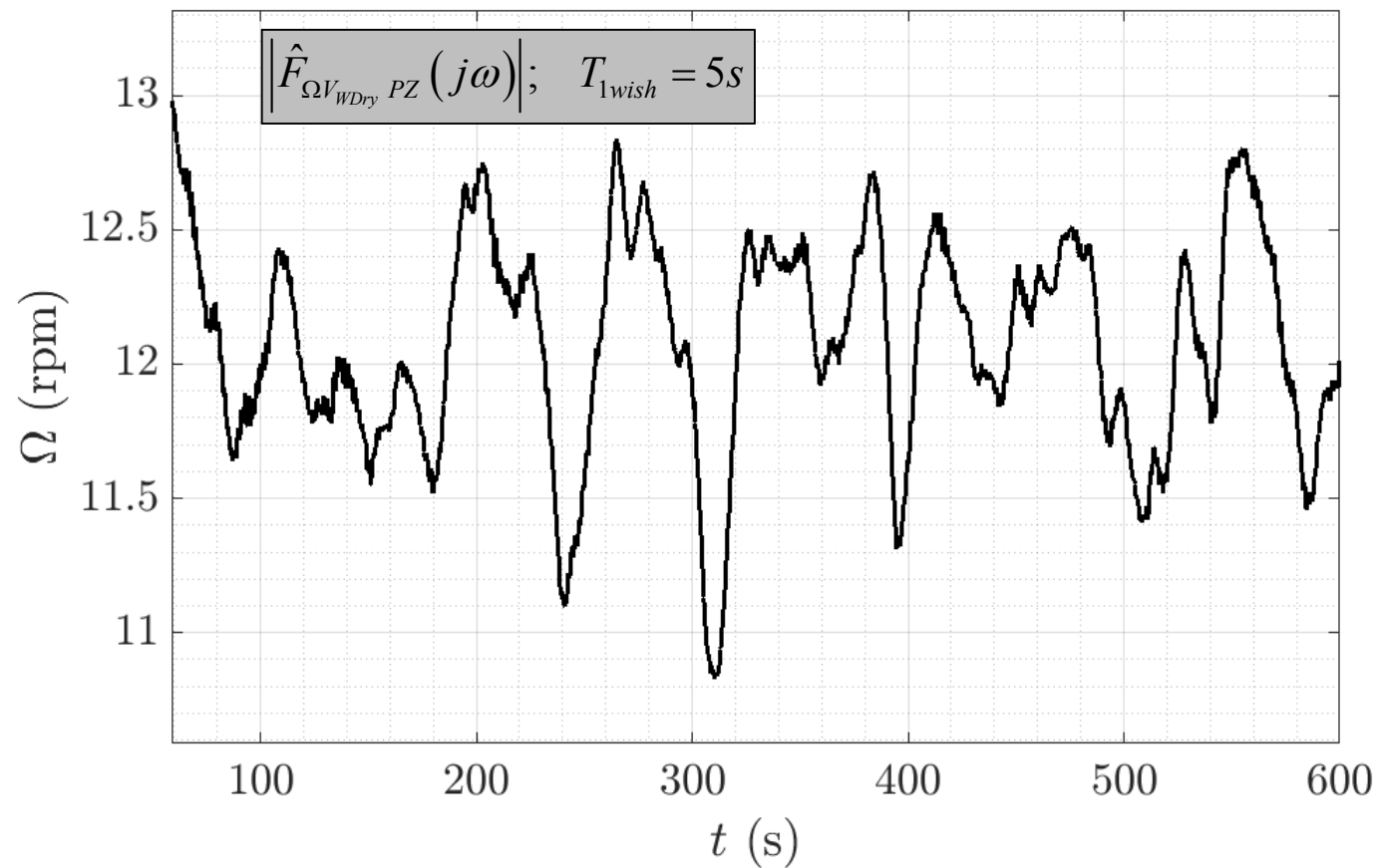
Nonlinear simulation (FAST)

zero/pole cancellation disturbance reaction (frequency domain)



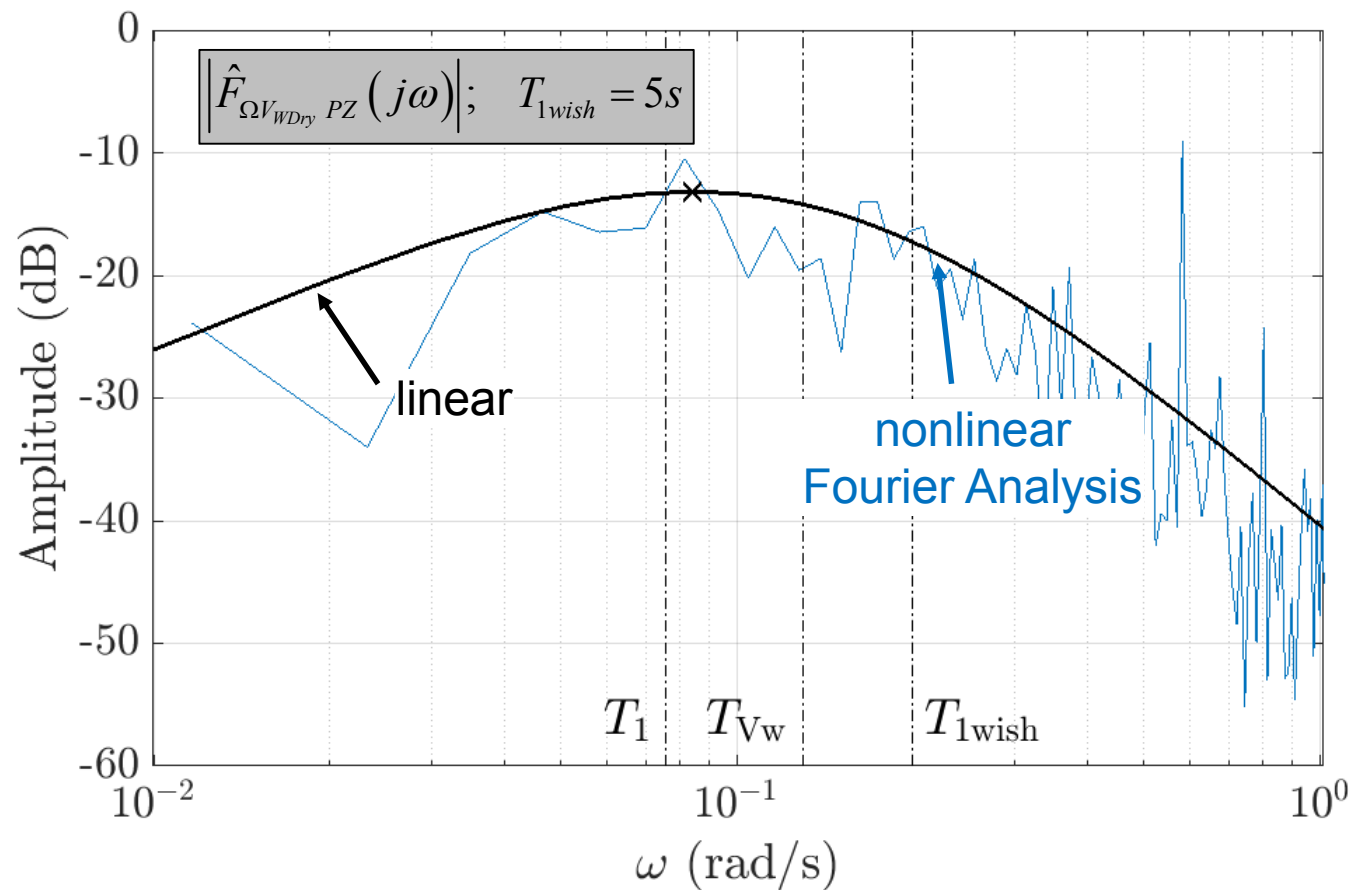
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Summary

- 2 analytical tuning methods for rotational speed Ω set-point tracking of variable-pitch wind turbines were shown
- Discussed by two criteria in the time- (step gust) and frequency-domain (1-D Dryden turbulence spectrum)
- Time and frequency domain results are hard to compare
- Give analytical advice to find a good trade off
- Deviate requirements for actuators
- Simple task (analytical tuning of a PI controller for a first order System) results in ambitious equations → further discussion of the equations and simplifications for a rapid “paper and pencil” controller predesign
- Equations for additional criteria like robustness criteria (phase margin, ...)



Thank you for your attention!

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